

Amendments To The Specification:

Please amend paragraph [001] as follows:

[001] This application is the US National Stage of International Application No. PCT/EP2004/004175, filed April 20, 2004 and claims the benefit thereof. The International Application claims the benefits of European Patent applications application No. 03011741.0 EP filed May 23, 2003, all of the applications are incorporated by reference herein in their entirety.

Please amend paragraph [010] as follows:

[010] Owing to the fact that a restrictor is interposed according to the invention, the hydraulic medium displaced by the individual pistons is first of all directed through the restrictor, a factor which advantageously leads to a reduction in the kinetic energy and to a comparatively slow displacement of the rotor. The loads acting on the bearing body can thus be reduced, whereby the risk of overloading is minimized. Even at a maximum force acting on the rotor, kinetic energy can be sufficiently dissipated by the restrictor arranged between the hydraulic piston arrangement and hydraulic system, so that overloading of the bearing as a result of dynamic forces of the rotor is prevented. Reliable mounting of the rotor of the gas turbine is thus ensured even during any possible occurrence of high dynamic thrusts.

Please amend paragraph [012] as follows:

[012] In an advantageous development, the bearing can additionally have at least one flow-control valve, designed as a restrictor, between the hydraulic piston arrangement and hydraulic system. This protection likewise increases the safety of the entire system and in addition makes it possible for the flow velocity of the hydraulic medium and thus the displacement speed of the rotor to be set.

Please amend paragraph [013] as follows:

[013] Furthermore, in an advantageous development, the bearing can have at least one flow-control valve, designed as a restrictor, between the hydraulic piston arrangement and hydraulic system. This protection likewise increases the safety of the entire system and in addition makes it possible for the flow velocity of the hydraulic medium and thus the displacement speed of the rotor to be set.

Please amend paragraph [018] as follows:

[018] According to a further feature of the invention, two hydraulic piston arrangements formed separately from one another are provided and are arranged opposite one another on the bearing body. In this configuration of the bearing according to the invention, the bearing body has a total of two hydraulic piston arrangements, which, depending on the configuration, have in each case a plurality of pistons. The pistons of the first hydraulic piston arrangement interact with a first stop surface and the pistons of the second hydraulic piston arrangement interact with a second stop surface. During a displacement of the rotor of the gas turbine, the pistons of the one hydraulic piston arrangement are extended as a result of this arrangement described above, whereas the pistons of the other hydraulic piston arrangement are retracted. During a displacement of the rotor in the opposite direction, a piston displacement of the hydraulic piston arrangement is likewise effected in the opposite direction. With respect to the thrust direction of the rotor, the one hydraulic piston arrangement is designated as a main track bearing and the other hydraulic piston arrangement is designated as a secondary track bearing.